

# PURLIN ASSEMBLY CLASS 8 NUTS

Hot Dip Galvanise  
Zinc Yellow Passivate  
Plain



## FASCIA HEX BOLT & NUT KIT CLASS 8.8 / CLASS 8 HOBSON STANDARD (ASSEMBLED)

Part	Size	Length (mm)	Pack	Pack Wgt (Kg)	Pallet	Stock HDG
KBAHTGCM100025	M10	25	200	7.37	16,000	✓
KBAHTGCM120030	M12	30	150	9.00	12,000	✓

*For further technical  
Information please contact  
Southeast Fasteners direct*



## METRIC HEX LOCK NUT CLASS 8 AS1112.4

Part	Size	Pack	Pack Wgt (Kg)	Stock HDG
NL08GCM12	M12	100	0.93	✓
NL08GCM16	M16	100	2.14	✓

? = G: Hot Dip Galvanise (HDG)    Z: Zinc Plate (Z/P)    Y: Zinc Yellow Passivate (ZYP)    P: Plain (PLN)



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# HOBSON 8.8 FASCIA BOLT ASSEMBLIES

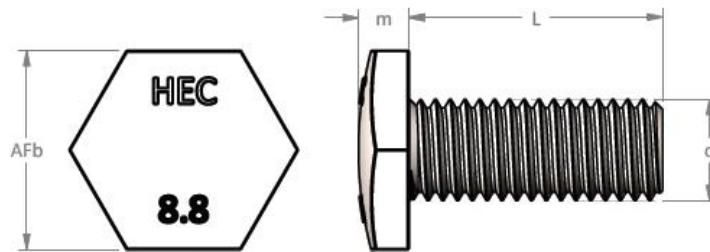
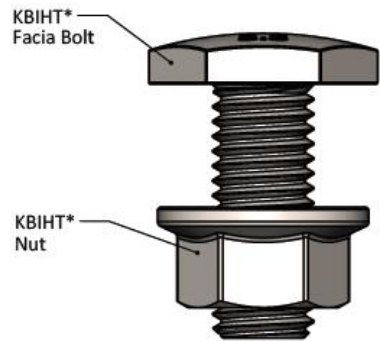


A Hobson 8.8 Fascia Bolt assembly consists of a property class 8.8 bolt and a class 8 nut. They come in two types of coating, zinc plated (ZP) and hot dip galvanised (HDG).

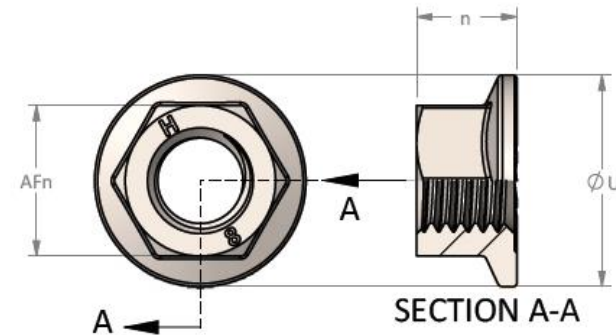
In the absence of tightening torque information from specifying engineers or fascia supplier, the indicative tightening torque shown below can be used as a guide to establish the suitable tightening torque.



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KBIHT\* Bolt



KBIHT\* Nut



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Part Number	Finish	Thread Size  d	Bolt			Nut			Indicative Tightening Torque <sup>1</sup>  T (N-m)	Bolt Tension <sup>2</sup>  P (N)
			Across Flats  AF <sub>b</sub> (mm)	Head Height  m (mm)	Bolt Length  L (mm)	Across Flats on Nut  AF <sub>n</sub> (mm)	Nut height  n (mm)	Flange Diameter  ØU (mm)		
KBIHTGCM100025	HDG	M10	22	4.9	25	16	10	22	42.0	16,850
KBIHTGCM120030	HDG	M12	24	5.5	30	18	12	26	73.0	24,450



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## Important Notes:

<sup>1</sup> Tightening torque  $T$  is calculated by using the basic formula,  $T = P \cdot k \cdot D$ , where  $P$  is the intended bolt tension assumed to be 50% percent of the bolt's proof load,  $k$  is the torque-friction factor and  $D$  is the thread diameter. The  $k$  value used for zinc plated and hot dip galvanised assemblies are 0.22 and 0.25 respectively. Note that the value of  $k$  can vary depending on thread conditions, thread/bearing surfaces lubrication and site conditions. All relevant bearing surfaces are assumed to be in full contact as shown in Fig. 1. The required bolt tension and torque should be validated/defined by the deciding engineer.

<sup>2</sup> Bolt tension is calculated at 50% percent of the bolt's proof load.

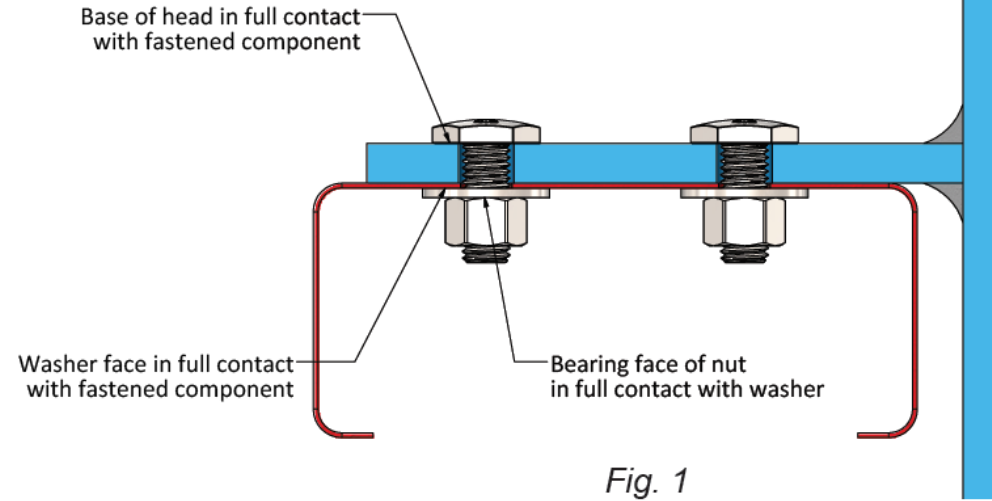


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## Installation Reminder:

Skewed bolt assembly orientation should be avoided. The base of the head and the base of the nut should be in full contact with the fastened component(s) as shown on Fig. 1.

Hole size and dimensions should be in accordance with AS4600 or as specified by the designing engineer.



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